

Quasi-normal modes and the resonant response of open phononic systems

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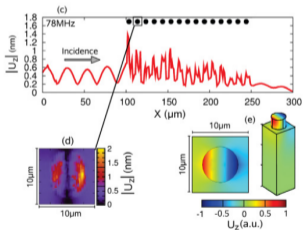


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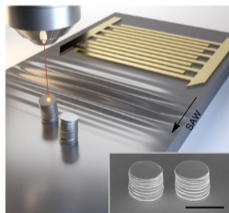
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2. Quasinormal modes
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4. Analysis of the response: poles, Q , modal volume
5. Acoustic reciprocity
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Physical motivation

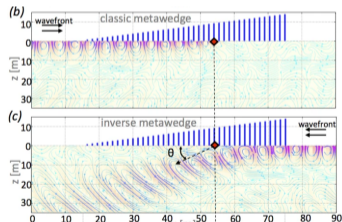
- There are many systems in phononics that deal with discrete resonators in interaction with a radiation substrate
- Following ideas from photonics/plasmonics, can we use the concept of quasi-normal mode (QNM) to predict their response?



Achaoui et al., PRB 2011
 10.1103/PhysRevB.83.104201



Raguin et al., Nat. Commun. 2019
 10.1038/s41467-019-12492-z



Colombi et al., Sci Reports 2016
 10.1038/srep27717

A short nomenclature of modes in wave physics

- A **closed** finite resonator sustains **normal modes**
- An **open** finite resonator sustains **quasi-normal modes**
 - If there is coupling to radiation modes, we speak of a **resonance**,
 - If there is no coupling, then we speak of **bound states in the continuum (BIC)**. BIC is lossless (real eigenfrequency) and often originates from a symmetry different from continuum waves
- If there is an invariance axis, a waveguide can be formed [examples: optical waveguides, optical fibers, surface acoustic waves, Rayleigh / Love waves in geophysics]
 - Outside the light or sound cone (where there are no propagating bulk waves), there are (evanescent) **guided waves**,
 - Inside the cone, there are either resonances (also termed **leaky waves**) or BICs!
- The same concepts and terms are used for **crystal waveguides** (that are periodic along a given direction)

Normal modes

- Normal modes are eigenmodes of *closed* structures
- Mathematically, for elastic waves they are the eigensolutions of

$$\omega_n^2 \rho(\omega_n) \mathbf{u}_n = -\nabla \cdot (\mathbf{c}(\omega_n) : \nabla \mathbf{u}_n) \quad (1)$$

with exterior boundary conditions (typically free or clamped)

- In the absence of loss, eigenfrequencies ω_n are real and eigenvectors \mathbf{u}_n are orthogonal
- The total energy of normal modes is bounded

$$H(\mathbf{u}_n) = \frac{1}{2} \left(\int S_n^* : \mathbf{c}(\omega) : S_n + \omega_n^2 \int \mathbf{u}_n^* \cdot \rho(\omega) \mathbf{u}_n \right) < \infty \quad (2)$$

$$S_n = \nabla \mathbf{u}_n \quad (3)$$

Quasinormal modes (QNMs)

- Quasi-normal modes (QNMs) are eigenmodes of *open* structures, suffering radiation to infinity
- They satisfy the same equation as normal modes

$$\omega_n^2 \rho(\omega_n) \mathbf{u}_n = -\nabla \cdot (\mathbf{c}(\omega_n) : \nabla \mathbf{u}_n) \quad (4)$$

but with outgoing-wave boundary conditions at infinity and with $\omega_n \in \mathbb{C}$

- Quality factor: $Q_n = \frac{\Re \omega_n}{2\Im \omega_n}$
- Their total energy is unbounded (similar to leaky modes of waveguides)

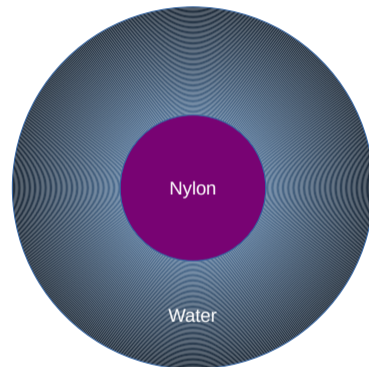
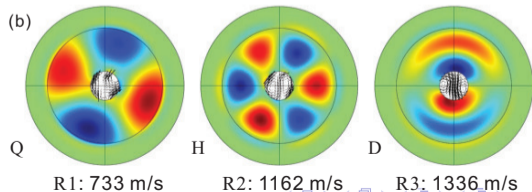
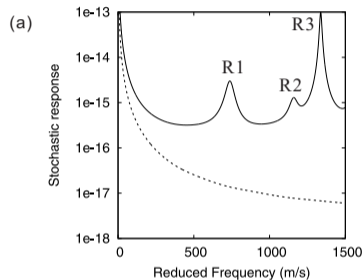


Figure: A nylon rod immersed in (infinite) water. What are its mechanical eigenmodes?

Response function and poles

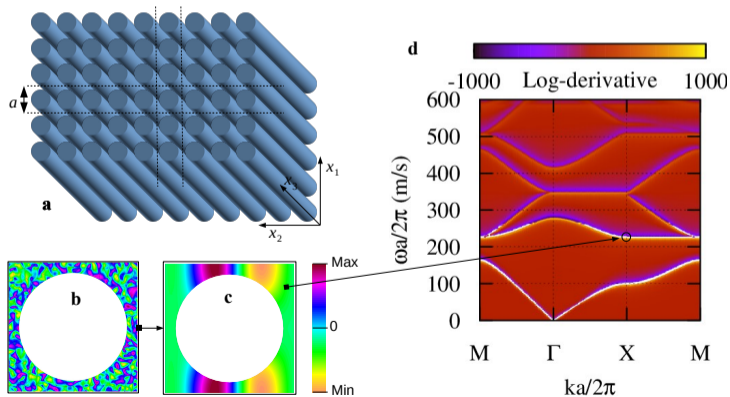
- We can also solve for the frequency response (with the PML) and observe poles
[Laude & Korotyaeva, Phys. Rev. B **97**, 224110 \(2018\)](#)
- Maxima give approximately the real part of the eigenfrequency
- The peak width gives a quality factor Q
- Resonant response gives approximately the QNM!
- Example: [Y. F. Wang, S. Y. Zhang, Y. S. Wang, & V. Laude, Phys. Rev. B **102** 144303 \(2020\)](#)



Stochastic excitation technique to obtain the resolvent band structure

- Physical idea: local excitations of the system result in resonances. Spatially random excitation guarantees none are forgotten

Laude & Korotyaeva, *Phys. Rev. B* **97**, 224110 (2018)



How can we obtain QNMs? There are 2 basic ways...

- 1 The beautiful and heroic way: Green's function techniques
 - Use some approximation in the finite region (e.g. FEM), couple it to the exterior analytic solution (plane waves in 1D, Bessel/Hankel functions in 2D, spheroidal harmonics in 3D)
 - Example: [M. B. Doost, W. Langbein, and E. A. Muljarov. "Resonant-state expansion applied to three-dimensional open optical systems." Physical Review A **90** 013834 \(2014\).](#)
- 2 The 'ugly' and efficient way: replace the infinite radiation medium with a perfectly matched layer (PML)
 - Use some eigenvalue solver to obtain the complex eigenmodes of the now 'closed' system
 - Example: [C. Sauvan, J. P. Hugonin, I. S. Maksymov, and P. Lalanne. Theory of the spontaneous optical emission of nanosize photonic and plasmon resonators. Physical Review Letters **110** 237401 \(2013\).](#)

Beware, it is not as easy as it seems...

PMLs have many 'spurious' eigenmodes, it is difficult to sort QNMs from them!

Definition of computational domains

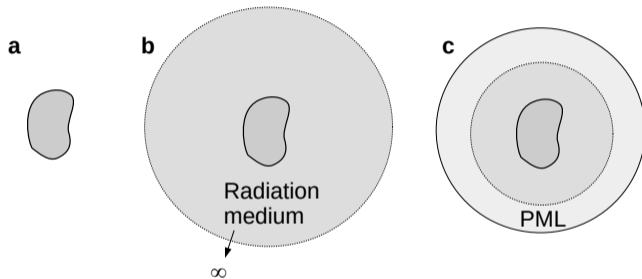


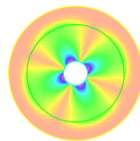
Figure: Definition of supporting domains for wave resonance and propagation. **(a)** Finite, closed domain supports normal modes. **(b)** Infinite, open domain supports quasinormal modes. **(c)** These can be approximated by closing the domain of computation with a perfectly matched layer (PML), that is the truncated image of the infinite domain in **(b)** in a complex coordinate transformation.

A practical algorithm to obtain one QNM

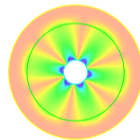
Start with ω_0 close to a maximum of the response

- 1 Initialization is stochastic: solve $(K - \omega_0^2 M)u_0 = F$ for random F
- 2 Iteratively solve $(K - \omega_n^2 M)u_{n+1} = Mu_n$; the solution converges to the nearest eigenvector
- 3 At the end of the iteration, normalize the eigenvector by $|u_n|_\infty$
- 4 and then evaluate $\omega_n^2 = u_n \cdot K \cdot u_n / u_n \cdot M \cdot u_n$

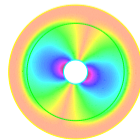
mode 1



mode 2



mode 3



Transposing Sauvan's method to elastodynamics – 1

[Sauvan et al. *Physical Review Letters* **110** 237401 (2013)]

Consider two different solutions of the elastodynamic equation ($\alpha = 1, 2$):

$$F_\alpha + \nabla \cdot (c : \nabla u_\alpha) = \frac{\partial}{\partial t} \left(\rho \frac{\partial u_\alpha}{\partial t} \right) \quad (5)$$

Weak form of the equation for one solution with the test function as the other solution:

$$\int S_2 : c(\omega_1) : S_1 - \omega_1^2 \int u_2 \cdot \rho(\omega_1) u_1 = \int u_2 \cdot F_1 \quad (6)$$

and the same equation with indices 1 and 2 permuted. Their difference leads to

$$\int S_2 : [c(\omega_1) - c(\omega_2)] : S_1 - \int u_2 \cdot [\omega_1^2 \rho(\omega_1) - \omega_2^2 \rho(\omega_2)] u_1 = \int u_2 \cdot F_1 - u_1 \cdot F_2 \quad (7)$$

Transposing Sauvan's method to elastodynamics – 2

Take sol. 2 as QNM number n and sol. 1 as the forced elastodynamic solution at ω :

$$\int S_n : [c(\omega) - c(\omega_n)] : S - \int \mathbf{u}_n \cdot [\omega^2 \rho(\omega) - \omega_n^2 \rho(\omega_n)] \mathbf{u} = \int \mathbf{u}_n \cdot \mathbf{F}, \forall n \quad (8)$$

The QNMs constitute a basis for the solution (eigenfunction expansion theorem):

$$\mathbf{u}(\omega) = \sum_m \alpha_m(\omega) \mathbf{u}_m \quad (9)$$

$$\sum_m B_{nm}(\omega) \alpha_m(\omega) = \int \mathbf{u}_n \cdot \mathbf{F} = F_n, \forall n \quad (10)$$

$$B_{nm}(\omega) = \int S_n : [c(\omega) - c(\omega_n)] : S_m - \int \mathbf{u}_n \cdot [\omega^2 \rho(\omega) - \omega_n^2 \rho(\omega_n)] \mathbf{u}_m \quad (11)$$

If the QNMs are known, the $B_{nm}(\omega)$ coefficients are easily computed, and the $\alpha_m(\omega)$ are obtained by solving a linear problem as a function of frequency.

An elliptical nylon rod in water

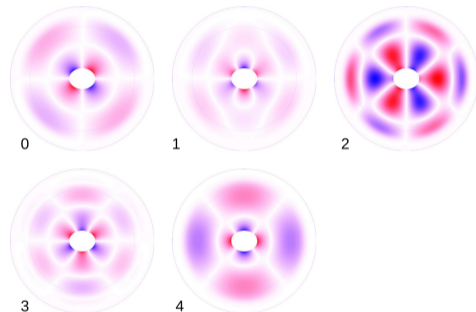
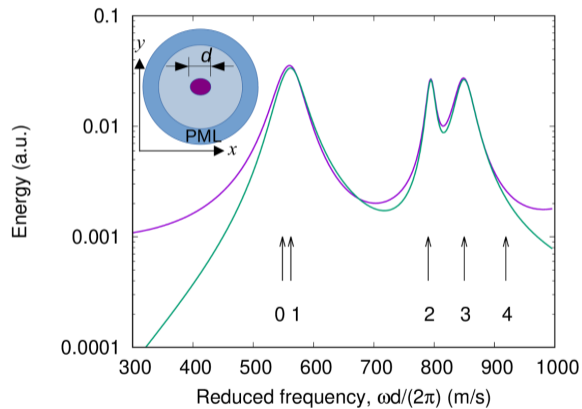


TABLE I. Characteristics for the QNMs of a cylindrical nylon rod immersed in water. The reduced frequency is $\omega d/(2\pi)$ with d the diameter of the rod.

Mode	0	1	2	3	4
Reduced frequency (m/s)	548	562	790	850	919
Q	12	48	24	30	12

Poling the response using QNMs

Sauvan's trick is to pole the previous equation by defining

$$A_{nm}(\omega) = \frac{1}{\omega - \omega_m} B_{nm}(\omega) \quad (12)$$

$$= \frac{1}{\omega - \omega_m} \left[\int S_n : [c(\omega) - c(\omega_n)] : S_m - \int \mathbf{u}_n \cdot [\omega^2 \rho(\omega) - \omega_n^2 \rho(\omega_n)] \mathbf{u}_m \right] \quad (13)$$

$A_{nm}(\omega_n) = 0$ if $m \neq n$ and

$$A_{nn}(\omega_n) = \int S_n : \frac{\partial c}{\partial \omega}(\omega_n) : S_n - \int \mathbf{u}_n \cdot \frac{\partial(\omega^2 \rho(\omega))}{\partial \omega}(\omega_n) \mathbf{u}_n \quad (14)$$

In the viscoelastic case (μ is the phonon viscosity tensor):

$$A_{nn}(\omega_n) = -2\omega_n \int \mathbf{u}_n \cdot \rho \mathbf{u}_n + \imath \int S_n : \mu : S_n \quad (15)$$

In the vicinity of each QNM:

$$\alpha_n(\omega) \approx \frac{1}{\omega - \omega_n} \frac{F_n}{A_{nn}(\omega_n)} + \Sigma_n(\omega) \quad (16)$$

Complex modal volume for QNMs

Considering some point in space r_0 , the modal volume is

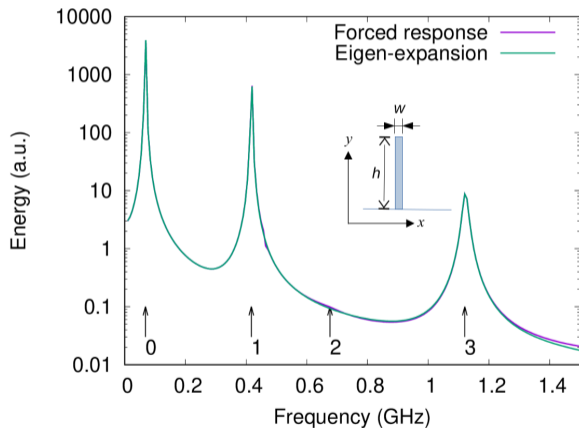
$$V_n = \frac{A_{nn}(\omega_n)}{2\omega_n[\rho(r_0)u_n^2(r_0)]} \quad (17)$$

$$r_0 = \underset{r}{\operatorname{argmax}} |\rho(r)u_n^2(r)| \quad (18)$$

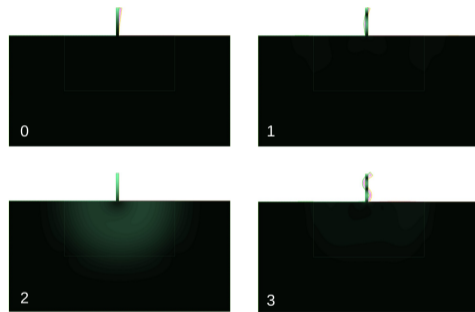
The modal volume is insensitive to multiplication by an arbitrary complex number. Indeed, QNMs are defined up to a complex multiplication constant only, as all eigenmodes. At resonance, $\omega \approx \Re\omega_n$ and the 'phononic Purcell effect' is

$$u(\Re\omega_n) \approx -i \frac{1}{\omega_n \Re\omega_n [\rho(r_0)u_n^2(r_0)]} \frac{Q_n}{V_n} F_n u_n. \quad (19)$$

A nanoscale nickel strip on fused silica ($F_x = 1$)

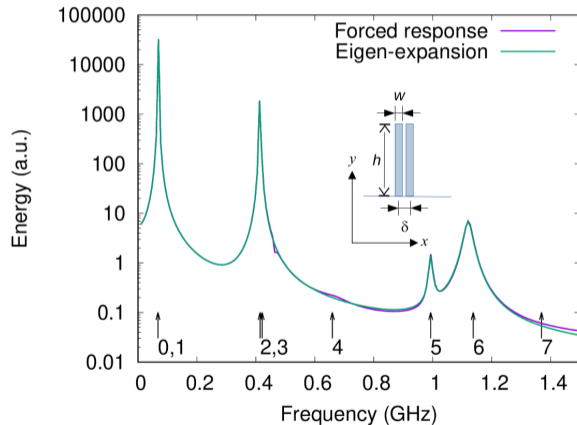


$w = 100$ nm, $h = 1000$ nm, 2D (strip) geometry

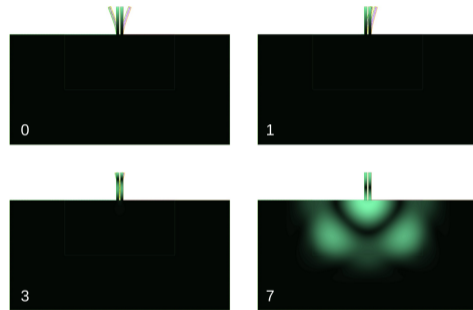


Mode	freq. (GHz)	Q	volume (μm^2)	polarization
0	0.0681	12600	0.0272	(0.997, 0.003, 0)
1	0.417	1000	0.0277	(0.979, 0.021, 0)
2	0.676	9	0.0923	(0, 0, 1)
3	1.12	60	0.0285	(0.959, 0.041, 0)

A nanoscale nickel tuning fork on fused silica ($F_x = 1$)



$w = 100$ nm, $h = 1000$ nm, $\delta = 50$ nm



Mode	freq. (GHz)	Q	volume (μm^2)	polarization
0	0.0674	240000	0.0536	(0.997, 0.003, 0)
1	0.0689	6700	0.0533	(0.997, 0.003, 0)
2	0.414	334	0.0569	(0.979, 0.021, 0)
3	0.421	4200	0.0524	(0.972, 0.028, 0)
4	0.659	3200	0.113	(0, 0, 1)
5	0.993	82	0.103	(0.01, 0.99, 0)
6	1.137	200	0.0365	(0.957, 0.043, 0)
7	1.369	10	0.474	(0, 0, 1)

Transposing Sauvan's method to acoustic waves – 1

The acoustic equation at frequency ω replacing the elastodynamic equation is

$$-\nabla \cdot (\rho^{-1} \nabla p) - \omega^2 B^{-1} p = \nabla \cdot (\rho^{-1} \mathbf{F}) = g \quad (20)$$

for pressure field $p(\mathbf{r})$ (a scalar field) and body force $\mathbf{F}(\mathbf{r})$. $B(\mathbf{r})$ is the elastic modulus and can be dispersive. The scalar source field $g(\mathbf{r})$ is introduced for convenience.

$$\int \nabla p_2 \rho_1^{-1} \nabla p_1 - \omega_1^2 \int p_2 B_1^{-1} p_1 = \int p_2 g_1 \quad (21)$$

with $\rho_1^{-1} = \rho^{-1}(\omega_1)$ and $B_1^{-1} = B^{-1}(\omega_1)$.

$$\int \nabla p_2 [\rho_1^{-1} - \rho_2^{-1}] \nabla p_1 - \int p_2 [\omega_1^2 B_1^{-1} - \omega_2^2 B_2^{-1}] p_1 = \int p_2 g_1 - p_1 g_2. \quad (22)$$

Transposing Sauvan's method to acoustic waves – 2

Take solution 2 as QNM number n and solution 1 as the current solution p depending on ω ,

$$\int \nabla p_n [\rho^{-1}(\omega) - \rho_n^{-1}] \nabla p - \int p_n [\omega^2 B^{-1}(\omega) - \omega_n^2 B_n^{-1}] p = \int p_n g, \forall n. \quad (23)$$

The QNMs constitute a basis for the solution (per the eigenfunction expansion theorem),

$$p(\omega) = \sum_m \beta_m(\omega) p_m. \quad (24)$$

$$\sum_m D_{nm}(\omega) \beta_m(\omega) = \int p_n g = g_n, \forall n \quad (25)$$

with

$$D_{nm}(\omega) = \int \nabla p_n [\rho^{-1}(\omega) - \rho_n^{-1}] \nabla p_m - \int p_n [\omega^2 B^{-1}(\omega) - \omega_n^2 B_n^{-1}] p_m. \quad (26)$$

If the QNMs are known, the $D_{nm}(\omega)$ coefficients are easily computed, and the $\beta_m(\omega)$ are obtained by solving a small linear problem as a function of frequency.

Transposing Sauvan's method to acousto-elastic resonances

$$\omega^2 \int_{\Omega_e} \mathbf{v} \cdot \rho_e \mathbf{u} - \int_{\Omega_e} \mathbf{S}(\mathbf{v}) : \mathbf{c} : \mathbf{S}(\mathbf{u}) - \int_{\sigma_e} v_n p - \omega^{-2} \int_{\Omega_a} \nabla q \rho_a^{-1} \nabla p + \int_{\Omega_a} q B^{-1} p + \int_{\sigma_a} u_n q = \int_{\Omega_e} \mathbf{v} \cdot \mathbf{F} \quad (27)$$

$$B_{nm}(\omega) = \int_{\Omega_e} \mathbf{S}_n : [\mathbf{c}(\omega) - \mathbf{c}(\omega_n)] : \mathbf{S}_m - \int_{\Omega_e} \mathbf{u}_n \cdot [\omega^2 \rho(\omega) - \omega_n^2 \rho(\omega_n)] \mathbf{u}_m \\ + \int_{\Omega_a} \nabla p_n [\omega^{-2} \rho^{-1}(\omega) - \omega_n^{-2} \rho_n^{-1}] \nabla p_m - \int_{\Omega_a} p_n [B^{-1}(\omega) - B_n^{-1}] p_m. \quad (28)$$

$$\mathbf{u}(\omega) = \sum_m \alpha_m(\omega) \mathbf{u}_m, \quad p(\omega) = \sum_m \alpha_m(\omega) p_m \quad (29)$$

Coefficients $\alpha_m(\omega)$ are obtained by solving

$$\sum_m B_{nm}(\omega) \alpha_m(\omega) = \int_{\Omega_e} \mathbf{u}_n \cdot \mathbf{F} = F_n, \forall n. \quad (30)$$

Conclusion and outlook

- Quasi-normal modes are the eigenmodes of resonant (open) structures
- Their eigenfrequencies can be obtained in the complex plane; the modal shape is obtained in the process
- Material dispersion and loss are accounted for
- Their modal volume is complex-valued!
- Only a small number of QNMs are required to 'predict' the frequency response for any excitation
- Theory provided for elastodynamics, acoustics, and acousto-elasticity

V. Laude and Y.-F. Wang, *Phys. Rev. B* 107, 144301 (2023)

<https://doi.org/10.1103/PhysRevB.107.144301>

Thanks and advertisement

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